

EX1 center of pressure

$$F = \rho g \bar{y} A \sin \theta$$

$$y_{cp} = \bar{y} + \frac{I}{A \bar{y}}$$

$$M_{BL} = F (\alpha + d - \bar{y} + y_{cp})$$

EX2 orifice & jet flow

$$\frac{V}{T} = Q_{actual}$$

$$Q_{theo} = A * V_{theo}$$

$$V_{theo} = \sqrt{2gh} \quad \text{or} \quad \sqrt{2g \Delta h}$$

$$\frac{Q_{actual}}{Q_{theo}} = C_d$$

$$\frac{V_{actual}}{V_{theo}} = C_v$$

$$C_v = \frac{x}{2\sqrt{y}h}$$

$$y_{intercept} = \log (C_d * \sqrt{2g} * A_{orifice})$$



Lab Hydraulics sheet

Bashar Fandi Alomari

inclined gate

$$F = \rho g A \bar{y} \sin \alpha$$

$$y_{cp} = \bar{y} + \frac{I}{A \bar{y}}$$



EX3 venturi tube

$$V_1 = \sqrt{\frac{2g(h_1 - h_2)}{\left(\frac{A_1}{A_2}\right)^2 - 1}}$$

$$V_2 = \sqrt{\frac{2g(h_2 - h_3)}{\left(\frac{A_2}{A_1}\right)^2 - 1}}$$

$$V_{theo} * A = Q_{theo}$$

$$Q_{actual} = \frac{V}{T}$$

$$\frac{Q_{actual}}{Q_{theo}} = C_d$$

$$Re = \frac{VD}{\nu} \quad \text{or} \quad \frac{\rho VD}{\mu}$$

$Re < 2000$ Laminar

$Re > 4000$ Turbulent

$4000 > Re > 2000$ Transition

stagnation point velocity = 0

EX:4 Impact of water jet

$$\Rightarrow F_{\text{measured}} = \rho Q v$$

$$\Rightarrow F_{\text{calculated}} = (1 - \cos \theta) \rho V Q$$

$$\Rightarrow \text{Momentum} = \rho V Q$$

$$\rightarrow Q_{\text{actual}} = \frac{V}{T}$$

$$\rightarrow V_{\text{nozel}} = \frac{Q_{\text{actual}}}{A_{\text{nozel}}}$$

$$\rightarrow V = \sqrt{(V_{\text{nozel}})^2 - 2gS}$$

Cone = 120°

f-plate = 90°

hemisphere = 180°

EX:5 Head loss

$$\frac{P_1}{\rho g} + \frac{v_1^2}{2g} + z_1 = \frac{P_2}{\rho g} + \frac{v_2^2}{2g} + z_2 + h_L$$

$\xrightarrow{h_s}$ $\xrightarrow{h_v}$ $\xrightarrow{h_p}$

$$h_L = F \frac{L}{D} \frac{v^2}{2g} \quad , \quad K = \text{loss coefficient}$$

$$F = \frac{64}{Re} \rightarrow \text{Laminar}$$

$$F = 0.0055 \left[1 + \left(20000 \frac{K_s}{D} + \frac{10^5}{Re} \right)^{1/2} \right]$$

experimentally

$$\Delta h = h_L = \frac{\Delta P}{\rho g}$$

h_v & h_p are constant

Roughness coefficients



EX: 6

$$Q = \frac{1}{n} R^{2/3} S^{1/2} A \rightarrow \text{Manning}$$

$$Q = C R^{1/2} S^{1/2} A \rightarrow \text{Chezy}$$

$$Q = \sqrt{\frac{8g}{f}} R^{1/2} S^{1/2} A \rightarrow \text{Darcy}$$

$$R = \frac{A}{P}$$

$$n_{\text{avg}} = \frac{\sum P \cdot n^2}{\sum P}$$

$$C = \frac{R^{1/6}}{n} = \sqrt{\frac{8g}{f}}$$

$$n = \frac{1}{C} R^{1/6}$$

| super | sub |
|-----------|-----------|
| $Fr > 1$ | $Fr < 1$ |
| $h > h_c$ | $h < h_c$ |
| $v > v_c$ | $v < v_c$ |
| $S > S_c$ | $S < S_c$ |

$$\log R \quad y_{\text{int}} = -\log n$$

EX: 7 specific energy

Rect.

$$V_c = \sqrt{g h_c} = \frac{Q}{A_c}$$

$$h_c = \left(\frac{q^2}{g} \right)^{1/3}$$

$$q = Q/b$$

$$R_c = \frac{A_c}{P_c}$$

$$S_c = \frac{V_c^2}{C^2 R_c}$$

$$E_c = \underbrace{\frac{3}{2} h_c}_{\text{Rectangular}} = h_c + \frac{V_c^2}{2g}$$

$$E = h + \frac{V^2}{2g}$$

$$V = \frac{q}{h}$$

$$F = \frac{h^2 S}{2} + \frac{q^2}{2h}$$

EX:8 Hydraulic Jump



$$Fr = \frac{v}{\sqrt{g \times h}}$$

$$h_2 = \frac{h_1}{2} \left[-1 + \sqrt{1 + 8Fr_1^2} \right]$$

$$E = h + \frac{v^2}{2g}$$

$$\Delta E_{\text{exp}} = E_2 - E_1$$

$$\Delta E_{\text{theo}} = \frac{(h_2 - h_1)^3}{4h_2 \times h_1}$$

$$h_j = h_2 - h_1$$

EX:9 weirs

Rectangle

$$Q_{\text{theo}} = \frac{2}{3} b (2g)^{1/2} H^{3/2}$$

$$\frac{Q_{\text{actual}}}{Q_{\text{theo}}} = C_d$$

$$y_{\text{int}} = \log \left(\sqrt{2g} \times \frac{2}{3} \times b^* \right)^{C_d}$$

$$\begin{matrix} \log Q_{\text{act}} \\ \swarrow \\ \log H^N \end{matrix}$$

Triangle

$$Q_{\text{theo}} = \frac{8}{15} (2g)^{1/2} \tan \frac{\theta}{2} H^{5/2}$$

$$\frac{Q_{\text{actual}}}{Q_{\text{theo}}} = C_d$$



$$y_{\text{int}} = \log \left(C_d \times \frac{8}{15} \times \sqrt{2g} \times \tan \frac{\theta}{2} \right)$$